

Università degli Studi di Brescia
Dipartimento di Ingegneria dell'Informazione

Robust Fractional Control

An overview of the research activity carried on at the University of
Brescia

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Why Fractional Control?

extra degrees of freedom and capability of modeling a wider range of dynamics

PUSH AHEAD THE FUNDAMENTAL ROBUSTNESS/PERFORMANCE TRADE-OFF!

Warning!

The design is much more complex

- 1 Fractional PID control
 - Tuning rules
- 2 \mathcal{H}_∞ Optimal Control
 - A model-matching problem
- 3 \mathcal{H}_∞ Model-matching controller design
 - Optimal Controller
 - Robust stability
- 4 Dynamic inversion of fractional systems
 - Command signal design
- 5 Optimal feedback/feedforward control
 - Combined feedback/feedforward design
- 6 Conclusions

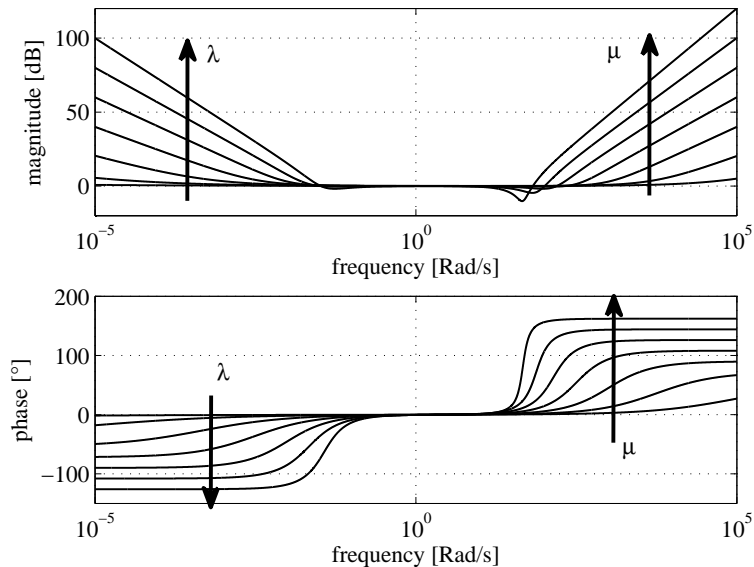
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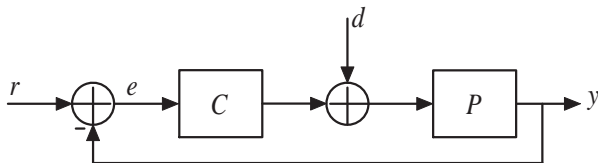
Fractional-Order Proportional-Integral-Derivative (FOPID) controllers are the natural generalization of standard PID controllers

FOPID controller

$$C(s) = K_p \frac{T_i s^\lambda + 1}{T_i s^\lambda} (T_d s^\mu + 1)$$

- λ and μ are the non integer orders of the integral and derivative terms
- they have 5 parameters instead of three
- they are more flexible: through the exponents a continuous regulations of the slope is possible...





Task

- set-point step following
- load disturbance step rejection

Dynamics

- Integral Plus Dead Time (IPDT) process $P(s) = \frac{K}{s} e^{-Ls}$
- First-Order Plus Dead Time (FOPDT) process $P(s) = \frac{K}{Ts+1} e^{-Ls}$
- Unstable First-Order Plus Dead Time (UFOPDT) process $P(s) = \frac{K}{1-Ts} e^{-Ls}$

Tuning rules: optimization function and constraints

In order to get optimal tuning rules the integrated absolute error (*IAE*) has been minimized.

$$IAE = \int_0^{\infty} |e(t)| dt = \int_0^{\infty} |r(t) - y(t)| dt,$$

Maximum Sensitivity M_s

$$M_s = \max_{\omega \in [0, +\infty)} \frac{1}{|1 + C(s)P(s)|}$$

M_s represents also the inverse of the minimum distance of the Nyquist plot from the critical point

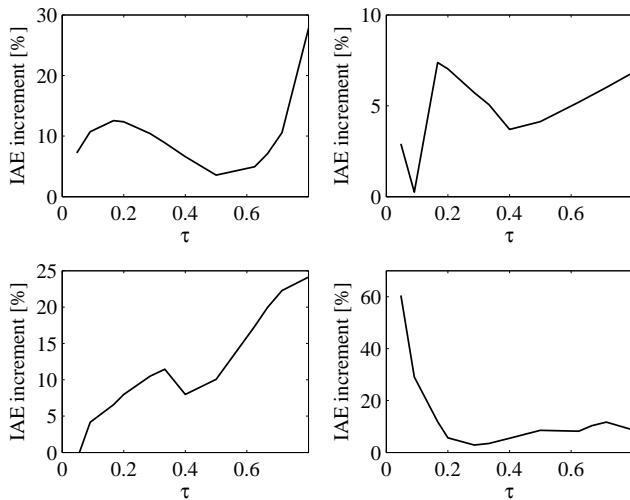
- $M_s = 1.4$ robust tuning
- $M_s = 2.0$ aggressive tuning

The optimization process has been numerically solved for different normalized dead time $\frac{L}{T}$

The results have been interpolated to obtain general tuning rules

The optimal *IAE* have been interpolated to obtain performance assessment rules

For the sake of comparison tuning rules have been developed also for integer PID controllers



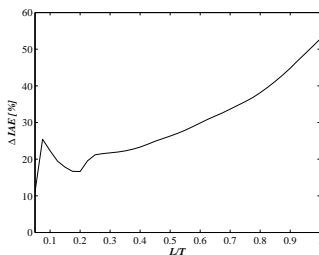
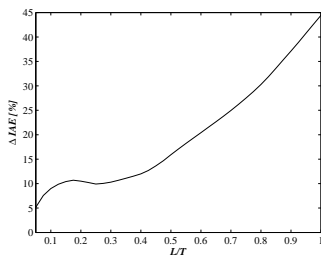
Top-left: set-point with $M_s = 1.4$. Top-right: set-point with $M_s = 2.0$. Bottom-left: load disturbance with $M_s = 1.4$. Bottom-right: load disturbance with $M_s = 2.0$.

Integral

M_s	1.4 sp	2.0 sp	1.4 ld	2.0 ld
ΔIAE [%]	17.2	6.34	19.1	22.7

The optimization can be performed just once!

Unstable



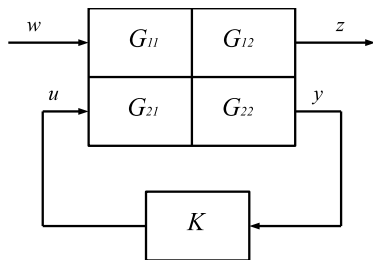
Left: set-point. Right: load disturbance

Fractional controllers always perform better than their integer counterparts!

- F. Padula and A. Visioli. *Tuning rules for optimal PID and fractional-order PID controllers*. Journal of Process Control, 21(1):69-81, 2011.
- F. Padula and A. Visioli. *Optimal tuning rules for proportional-integral-derivative and fractional-order proportional-integral-derivative controllers for integral and unstable processes*. IET Control Theory and Applications, 6(6):776-786, 2012.
- F. Padula and A. Visioli. *Set-point weight tuning rules for fractional-order PID controllers*. Asian Journal of Control, 15(4):1-13, 2013.

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The standard \mathcal{H}_∞ control problem



Let \mathcal{C} be the set of stabilizing controllers

Problem 1

$$\min_{K \in \mathcal{C}} \|T_{zw}\|_\infty$$

where

$$T_{zw} = G_{11} + G_{12}K(1 - G_{22}K)^{-1}G_{21}$$

A Model-matching problem

Theorem: Youla parametrization

The set \mathcal{C} of all stabilizing controllers K is:

$$\mathcal{C} = \left\{ \frac{X + MQ}{Y - NQ} : Q \in \mathcal{H}_\infty \right\}$$

where $P = MN^{-1}$ and M, N, X, Y satisfy the Bezout identity $NX + MY = 1$

using the previous result

$$z = (T_1 - QT_2)w$$

Problem 2

Find $Q \in \mathcal{H}_\infty$ such that the model-matching error $\|T_1 - QT_2\|_\infty$ is minimized, where both T_1 and T_2 are in \mathcal{FRH}_∞

using inner-outer factorization

$$\|T_1 - T_2Q\|_\infty = \|R - X\|_\infty$$

where

$$R \in \mathcal{FRL}_\infty, X \in \mathcal{H}_\infty$$

Nehari's theorem

There exists a closest \mathcal{H}_∞ -matrix X to a given \mathcal{L}_∞ -matrix R , and $\|R - X\| = \|\Gamma_R\|$, where Γ_R is the Hankel operator with symbol R

R can be factorized as $R = R_1 + R_2$ with $R_1 \in \mathcal{RL}_\infty$ (integer!) unstable and analytic in the left half plane (antistable) and $R_2 \in \mathcal{H}_\infty$ and it holds that $\Gamma_R = \Gamma_{R_1}$

R_1 is integer, thus Γ_R has finite rank and can be computed by means of known techniques

It can be shown that the optimal model-matching error is integer and real-rational...

...this is a Nevanlinna-Pick optimal interpolation problem!

Each RHP zero of T_2 plays the role of an interpolation constraint to avoid internal instability: $Q \in \mathcal{H}_\infty$, no zero/pole cancelations in the RHP

Theorem

Consider the model-matching problem, the optimal model matching error is an all-pass in \mathcal{RH}_∞ whose coefficients are completely determined by the interpolation constraints

$$E^o(z_i) = T_1(z_i) \quad i = 1, \dots, n$$
$$\left. \frac{d^k E^o(s)}{ds^k} \right|_{s=z_i} = \left. \frac{d^k T_1(s)}{ds^k} \right|_{s=z_i} \quad k = 1, \dots, m_i - 1; \quad i = 1, \dots, n$$

being m_i the multiplicity of the i th RHP zero of T_2 and E^o the optimal model-matching error

The optimal interpolation error is integer and real-rational!

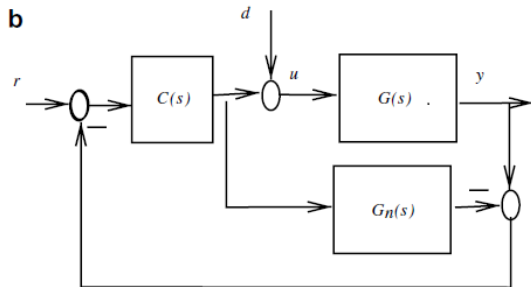
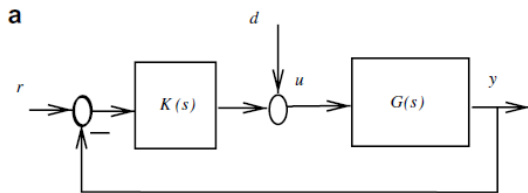
The optimal Youla parameter Q is \mathcal{FRH}_∞

The optimal controller is a fractional real-rational function

F. Padula, S. Alcantara, R. Vilanova, and A. Visioli. *\mathcal{H}_∞ control of fractional linear systems*. Automatica 49(17):2276-2280, 2013.

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Weighted model-matching problem



a) feedback configuration, b) IMC configuration

Using the IMC controller $C(s)$ the closed loop transfer function has a very simple expression:

$$T(s) = G_n(s)C(s)$$

the equivalent feedback controller can be easily recovered by means of:

$$K(s) = \frac{C(s)}{1 - C(s)G_n(s)}$$

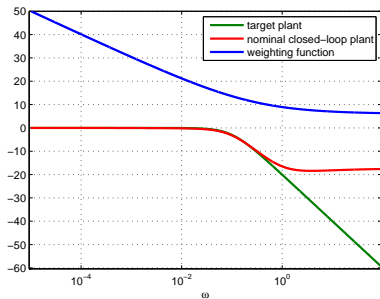
By means of the IMC controller we can set up a *weighted* model-matching problem:

$$C^o(s) := \min_{C(s)} \|W(s)(M(s) - C(s)G_n(s))\|_\infty$$

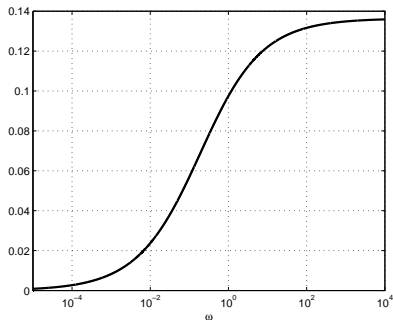
Find $C(s)$ such as the ∞ -norm of the *weighted* difference between the nominal closed-loop transfer function and the desired closed-loop transfer function is minimized

The role of the weighting function is of main concern: it allows the user to give more importance to certain frequency ranges (typically low frequencies) and less importance to other frequency ranges (typically high frequency).

Weighted model-matching problem



nominal closed-loop frequency response (red), target frequency response (green), weighting function (blue)



model-matching error

The model mismatch is bigger at those frequencies where the weighting function is smaller

The functions involved in the model-matching problem are chosen as follows:

- 1 Process model:

$$G_n(s) = \frac{K}{1 + Ts^\alpha} e^{-Ls}$$

- 2 Nominal process transfer function

$$G_n(s) = K \frac{1 - Ls}{1 + Ts^\alpha}$$

- 3 Target closed-loop transfer function

$$M(s) = \frac{1}{1 + T_m s^\lambda}$$

- 4 Weighting function

$$W(s) = \frac{1 + z s^\mu}{s^\mu}$$

where T_m , λ , z and μ are parameters to be selected

the equivalent feedback controller is

$$K^o(s) = \frac{1}{K} \frac{1 + Ts^\alpha}{\frac{\rho}{\gamma} s^\mu + T_m s^\lambda + T_m (z + \frac{\rho}{\gamma}) s^{\lambda+\mu}} \times \left(1 + \frac{\rho T_m}{\gamma L^\mu} s^\mu \right) \left(1 + \frac{-\sum_{k=m}^{n-1} L_n^k s^{\frac{k}{n}} + \sum_{k=n}^{m-1} L_n^k s^{\frac{k}{n}}}{\sum_{k=0}^{n-1} L_n^k s^{\frac{k}{n}}} \right)$$

it has the same low-frequency behavior of a filtered FOPID controller, neglecting the last term (that for low frequencies tends to one) a suboptimal controller is obtained:

$$\tilde{K}(s) = \frac{1}{K(\frac{\rho}{\gamma} + T_m)} \frac{(1 + Ts^\alpha)(1 + T_m \frac{L^\mu + z}{L^\mu + T_m} s^\mu)}{s^\mu (1 + T_m \frac{\frac{\rho}{\gamma} + z}{\frac{\rho}{\gamma} + T_m} s^\mu)}$$

It is a filtered FOPID controller in series form

$$\text{When } \mu = 1, \tilde{K}(s) = K^o(s)$$

In any case the suboptimal controller stabilizes the nominal system!

Assume that the process belongs to a family \mathfrak{F} defined as:

$$\mathfrak{F} = \{G(s) = G_{n_t}(s)(1 + \Delta_m(s)) : |\Delta_m(j\omega)| < |\Gamma(j\omega)|\}$$

$\Delta_m(s) = (G(s) - G_{n_t}(s))/G_{n_t}(s)$ is the uncertainty description

$\Gamma(j\omega)$ is a frequency dependent function that upper bounds the system uncertainty.

Robust stability condition:

$$\|\Gamma(s)T_n(s)\|_\infty < 1$$

where $T_n(s)$ is the nominal closed-loop transfer function.

Sufficient condition:

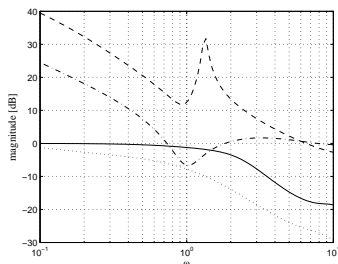
$$|T_n(j\omega)| < |1/\Gamma(j\omega)|$$

The right hand side of this inequality is usually a low-pass transfer function. **It defines a robust stability boundary.**

Based on robustness and desired bandwidth tuning guidelines have been provided.

$$G(s) = \frac{1}{s^2 + 0.4s + 1} e^{-0.6s}$$

$$G_{n_t,F}(s) = \frac{1}{1.05s^{1.702} + 1} e^{-0.74s} \quad G_{n_t,I}(s) = \frac{1}{0.566s + 1} e^{-0.9s}$$

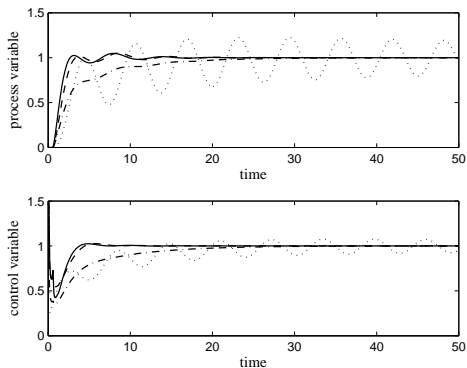


T_m has been fixed to 1.5 ($\mu = 1$)

In the fractional case z can be reduced to 0 preserving robust stability. The selection of z can be done just to speed up or slow down the system response

In the integer case it is necessary to set $z = 10$ to achieve robust stability

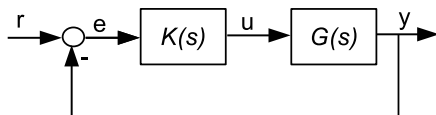
Step responses



Step-responses with $T_m = 1.5L$ and $\mu = 1$: integer model (dotted line $z = 10$) and the fractional model for different values of z (dash-dot line $z = 10$, dashed line $z = 1$ and solid line $z = 0.1$)

F. Padula, R. Vilanova, and A. Visioli. *\mathcal{H}_∞ optimization based fractional-order PID controllers design*. International journal of Robust and Nonlinear Control (in press, available online).

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where $G(s) = \bar{G}(s)e^{-Ls}$ and $T(s) = \frac{K(s)G(s)}{1+K(s)G(s)}$

Find a command signal such that a smooth transition of the output between 0 and 1 is obtained within in finite amount of time τ satisfying a set of constraints on the control variable and its derivatives

Given a sufficiently smooth desired output $\bar{y}(\cdot; \tau)$ find the command signal $r(\cdot; \tau)$ such that, for the τ -parameterized couple $r(\cdot; \tau), \bar{y}(\cdot; \tau)$, it holds that

$$\mathcal{L}[\bar{y}(t - L; \tau)] = T(s)\mathcal{L}[r(t; \tau)]$$

Moreover, satisfy

$$|D^i u(t; \tau)| < u_M^i, \forall t > 0, i = 0, 1, \dots, l$$

Desired output, τ -parameterized *transition polynomial* $\bar{y}(t; \tau) \in \mathbf{C}^{(n)}$

$$\bar{y}(t; \tau) := \begin{cases} 0 & \text{if } t < 0 \\ \frac{(2n+1)!}{n! \tau^{2n+1}} \sum_{r=0}^n \frac{(-1)^{n-r} \tau^r t^{2n-r+1}}{r!(n-r)!(2n-r+1)} & \text{if } 0 \leq t \leq \tau \\ 1 & \text{if } t > \tau \end{cases}$$

- smooth, through the parameter n the regularity of the transition polynomial can be arbitrarily selected;
- monotonic;
- finite time transition.

Consider the transfer function $H(s)$ of Σ :

$$H(s) = \frac{b(s)}{a(s)} = \frac{\sum_{k=0}^m b_k s^{k\nu}}{s^{p\nu} + \sum_{k=0}^{p-1} a_k s^{k\nu}}$$

where ν is the commensurate order.

$\rho = (p - m)\nu$ the relative order of Σ .

Define the set of all the cause/effect pairs associated with Σ :

$$\mathcal{B} := \left\{ (u(\cdot), y(\cdot)) \in P_c \times P_c : \sum_{k=0}^m b_k D^{k\nu} u = D^{\rho\nu} y + \sum_{k=0}^{p-1} a_k D^{k\nu} y \right\}$$

Consider the system $H(s)$, given the desired transition polynomial (i.e. $C^{(k)}$ for some $k \in \mathbb{N}$) $\bar{y}(t; \tau)$, find the input $u(t; \tau)$ such as the τ -parameterized couple $(u(\cdot; \tau), \bar{y}(\cdot; \tau)) \in \mathcal{B}$, $\bar{y}(0; \tau) = 0$ and $\bar{y}(t; \tau) = 1 \forall t \geq \tau$, moreover satisfy

$$|D^i u(t; \tau^*)| < u_M^i, \forall t > 0 \quad i = 0, 1, \dots, l$$

Using Laplace transform, the inversion is algebraic in the frequency domain:

$$U(s; \tau) = H^{-1}(s) \bar{Y}(s; \tau)$$

Σ is assumed to be commensurate here thus the following techniques apply:

- Polynomial division
- Partial fraction expansion

and the inverse system can be decomposed as follows

$$H^{-1}(s) = \gamma_{n-m} s^p + \gamma_{n-m-1} s^{p-\nu} + \dots + \gamma_1 s^\nu + \gamma_0 + H_0(s)$$

$H_0(s)$, zero dynamics of Σ , strictly proper

$$H_0(s) = \sum_{i=1}^m \frac{g_i}{(s^\nu - \lambda_i)^{k_i+1}}$$

Inverse transforming...

...the zero dynamics is the summation of Mittag-Leffler functions

$$\eta_0(t) = \sum_{i=1}^m \frac{g_i}{k_i!} \varepsilon_{k_i}(t, \lambda_i; \nu, \nu) = \sum_{i=1}^m \frac{g_i}{k_i!} t^{k_i \nu + \nu - 1} \frac{d_i^k}{d(\lambda_i t^\nu)^{k_i}} E_{\nu, \nu}(\lambda t^\nu)$$

Proposition

If $n > [\rho] + 1 + l$, for τ sufficiently large then

$$\begin{aligned} u(t; \tau) &= \gamma_{n-m} D^\rho \bar{y}(t; \tau) + \gamma_{n-m-1} D^{\rho-\nu} \bar{y}(t; \tau) + \dots \\ &\quad + \gamma_1 D^\nu \bar{y}(t; \tau) + \gamma_0 \bar{y}(t; \tau) + \int_0^t \eta_0(t-\xi) \bar{y}(\xi; \tau) d\xi \end{aligned}$$

The convolution integral becomes

$$\begin{aligned} \int_0^t \eta_0(t-\xi) y(\xi; \tau) d\xi &= \sum_{i=1}^m \frac{g_i}{k_i!} \left[\frac{(2n+1)!}{n! \tau^{2n+1}} \sum_{r=0}^n \frac{(-1)^{n-r} \tau^r}{r!(n-r)!(2n-r+1)} (2n-r+1)! \right] \\ &\times \left[\varepsilon_{k_i}(t, \lambda_i; \nu, 2n-r+2+\nu) \right. \\ &\quad - \left. \begin{cases} 0 & \text{if } 0 \leq t \leq \tau \\ \sum_{j=0}^{2n-r+1} \binom{2n-r+1}{j} (2n-r+1-j)! \tau^j \\ \quad \times \varepsilon_{k_i}(t-\tau, \lambda_i; \nu, 2n-r+2-j+\nu) & \text{if } t > \tau \end{cases} \right] \\ &+ \left[\begin{cases} 0 & \text{if } 0 \leq t \leq \tau \\ \varepsilon_{k_i}(t-\tau, \lambda_i; \nu, 1+\nu) & \text{if } t > \tau \end{cases} \right] \end{aligned}$$

The open loop system is first inverted obtaining $r_{ol}(t; \tau)$ via dynamic inversion of the delay-free open loop transfer function

$$K(s)\bar{G}(s)$$

a delayed correction term is then added to avoid the delayed feedback effect

$$r_c(t; \tau) = \bar{y}(t - L; \tau)$$

finally, the command signal is computed

$$r(t; \tau) = r_{ol}(t; \tau) + r_c(t; \tau)$$

under the existence condition

$$n \geq [\rho_{\bar{G}}] + 1 + l$$

F. Padula and A. Visioli. *Inversion-based feedforward and reference signal design for fractional constrained control systems*. *Automatica*. 50(8):2169-2178, 2014.

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So far we have introduced two results:

- robust controller design
- command signal synthesis

now we want to put the previous results together.

Consider the family of plants

$$\mathfrak{F} = \left\{ \tilde{G}(s) = \frac{\tilde{K}}{\tilde{T}s^{\tilde{\lambda}} + 1} e^{-\tilde{L}s} : \tilde{K} \in [K_{min}, K_{max}], \right. \\ \left. \tilde{T} \in [T_{min}, T_{max}], \tilde{\lambda} \in [\lambda_{min}, \lambda_{max}], \tilde{L} \in [L_{min}, L_{max}] \right\}$$

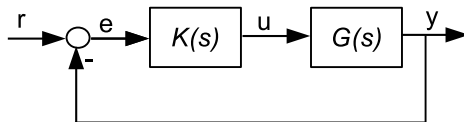
and the nominal system

$$G(s) = \frac{K}{Ts^{\lambda} + 1} e^{-Ls}$$

whose parameters are the mean values of the corresponding uncertainty intervals

Define extremal system $G_i(s)$ $i = 1, \dots, 16$ for the family \mathfrak{F} each system obtained with any possible combination of the extremal values of the uncertainty intervals.

Given a unity feedback loop



We want to design a controller $K(s)$ and a command signal $r(t)$ to satisfy:

- robustness
- control variable limitation
- overshoot limitation
- settling time minimization

for the whole family of plants \mathfrak{F}

define the worst-case settling time (at a given percentage)

$$t_{s,wc}(\tau, T_m, \mathbf{z}) := \max_{i=1,\dots,16} t_{s,i}(\tau, T_m, \mathbf{z}),$$

$\{\tau, T_m, \mathbf{z}\}$ is a set of tuning parameter

Min-max problem

$$\min_{\tau, T_m, \mathbf{z}} t_{s,wc}(\tau, T_m, \mathbf{z})$$

subject to

- 1 (Robust stability) $\|\Gamma(s)T_n(s)\|_\infty < 1$;
- 2 (Maximum overshoot) $\max y_i(t; \tau, T_m, \mathbf{z}) < y_f(1 + O_{max}), i = 1, \dots, 16$;
- 3 (Maximum control variable) $\max |u_i(t; \tau, T_m, \mathbf{z})| < U_{max}, i = 1, \dots, 16$;

where $(u_i(\cdot), y_i(\cdot))$ is the input-output couple for the i th extremal system, y_f is the set-point value, $O_{max} > 0$ is the maximum allowable overshoot and $U_{max} > 0$ the maximum acceptable control variable.

Optimal feedback/feedforward control

only a constraints on the control variable is imposed ($l = 0$)
the existence condition for both control signal and command signal reduces to

$$n \geq [\rho_{K\bar{G}}] + 1$$

moreover it can be shown

Lemma

There always exists a couple of parameters T_m, z such that the optimal controller $K^o(s)$ stabilizes the family \mathfrak{F} provided that the parametric uncertainty over the process dc-gain K is lower than 1, *i.e.*,

$$\frac{K_{max} - K}{K} < 1$$

Theorem

The min-max problem is solvable provided that

$$\frac{K_{max} - K}{K} < 1$$

and

$$U_{max} > \frac{y_f}{G_i(0)}, i = 1, \dots, 16$$

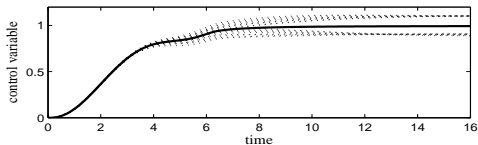
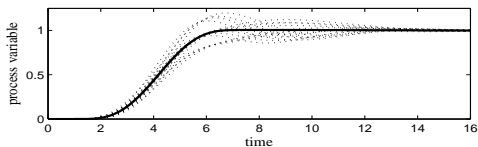
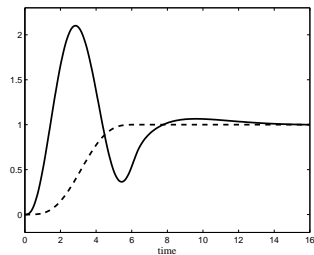
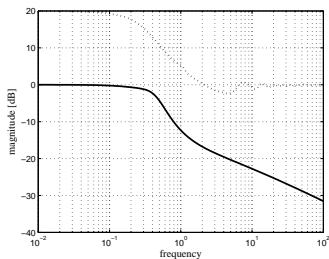
$$G(s) = \frac{1}{s^{1.5} + 1} e^{-s}$$

- 1 uncertainty of $\pm 10\%$ over the plant's parameters
- 2 settling time at 2%
- 3 unitary set-point value $y_f = 1$
- 4 maximum control variable of $U_{max} = 1.5$
- 5 maximum overshoot of $O_{max} = 0.2$

Solving procedure

- Numerically compute the robust stability boundary by gridding the process uncertainty
- Obtain $\Gamma(j\omega)$ by upper bounding the computed uncertainties for each frequency
- Select $\beta = \max[1, \lambda] = 1.5$ and a transition polynomial with regularity $n = 3$ to satisfy the existence condition
- Numerically (genetic algorithm) solve the min-max problem

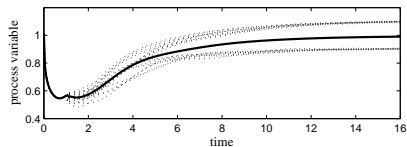
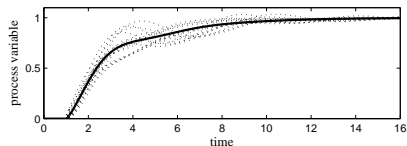
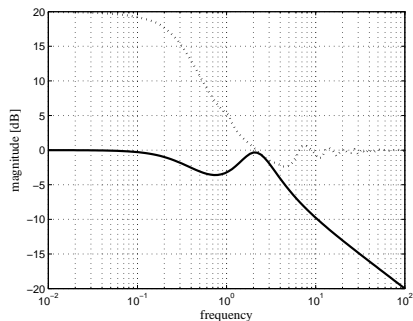
Example



The obtained optimal parameters are
 $T_m = 3.5469$, $z = 3.4155$
and $\tau = 6.3122$

the optimal worst-case settling time is $t_{s,wc} = 13.42$

Results optimizing the settling time but using a step command signal...



The obtained optimal parameters are $T_m = 4.8747$, $z = 5.6705$ and $\tau = 9.7502$

the optimal worst-case settling time is $t_{s,wc} = 18.11$

...the combined feedback/feedforward optimization performance improvements is 26 % !

Agenda

- 1 Fractional PID control
 - Tuning rules
- 2 \mathcal{H}_∞ Optimal Control
 - A model-matching problem
- 3 \mathcal{H}_∞ Model-matching controller design
 - Optimal Controller
 - Robust stability
- 4 Dynamic inversion of fractional systems
 - Command signal design
- 5 Optimal feedback/feedforward control
 - Combined feedback/feedforward design
- 6 Conclusions

- a complete set of constrained optimal time-scale invariant tuning rules for PID and FOPID controllers, for stable unstable and integral processes
- the solution of the scalar standard \mathcal{H}_∞ control problem for fractional SISO LTI systems
- an \mathcal{H}_∞ model-matching robust design methodology suitable for both monotonic and nonmonotonic dynamics
- the solution of a constrained optimal input-output dynamic inversion problem for fractional LTI systems
- an inversion-based feedforward signal design for fractional control loops
- a combined feedback/feedforward control design technique to cope with uncertainty in an effective way
- ...

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